

High frequency asymptotics of acoustical nucleate pool boiling noise spectrum in compressible viscous liquids

L. LIKHTEROV

Department of Mechanical Engineering, Ben Gurion University of the Negev, P.O.B. 653,
 Beer Sheva, 84105, Israel

(Received 3 April 1992 and in final form 29 April 1993)

Abstract—An expression for the high frequency asymptotics of the acoustic noise spectrum in nucleate pool boiling is derived taking into account the compressibility and viscosity effects. The expression is obtained in quasi-acoustical approximation. The theoretical results qualitatively agree with previous measurements of boiling noise.

INTRODUCTION

THE ACOUSTICAL emission from boiling bubbles has been studied experimentally and theoretically. These studies investigated the nucleate pool boiling which occurs when a heater is submerged in a tank of initially stagnant liquid. The spectra of acoustical noise in pool boiling were obtained [1–3], and it was established that the spectral density of boiling noise spectra varies as $S \sim f^4$ for low frequencies and $S \sim f^{-2}$ for high frequencies [4]. Relatively few detailed studies have been made for a theoretical definition of spectrum form. The works of Fitzpatrick [5], Benjamin [6], Gilmore [7] may be mentioned, in which they considered isothermal cases connected with cavitation problems, and Likhterov and Elperin [8] where the process of sound generation was treated for cases of bubble growth and implosion. It has been shown that the high frequency part of the noise spectrum varies as $S \sim f^{-2/5}$, but such slow decrease of the spectral density stems from assumption of liquid incompressibility and neglecting dissipative effects. The present work was undertaken to consider these factors and to extend the information available on the sound emission in the nucleate pool boiling.

THEORETICAL ASPECTS

The momentum equation for compressible and viscous liquid is

$$\frac{\partial \mathbf{V}}{\partial t} + (\mathbf{V} \text{ grad}) \mathbf{V} = -\frac{1}{\rho} \text{ grad } p + \frac{\nu}{3} \text{ grad div } \mathbf{V} + \nu \Delta \mathbf{V} \quad (1)$$

and continuity equation is

$$\frac{1}{\rho} \frac{d\rho}{dt} = -\text{div } \mathbf{V}, \quad (2)$$

where \mathbf{V} is velocity vector.

A problem under consideration has a spherical symmetry and consequently, velocities in all liquids have a potential ($r\phi$), therefore

$$\text{curl } \mathbf{V} = 0. \quad (3)$$

The last term in equation (1) can be represented as

$$\nu \Delta \mathbf{V} = \nu \text{ grad div } \mathbf{V} - \nu \text{ curl curl } \mathbf{V}$$

and because of equation (3), it is apparent that

$$\nu \Delta \mathbf{V} = \nu \text{ grad div } \mathbf{V}, \quad (4)$$

and Navier–Stokes equations get simplified:

$$\frac{\partial \mathbf{V}}{\partial t} + (\mathbf{V} \text{ grad}) \mathbf{V} = -\frac{1}{\rho} \text{ grad } p + \frac{4}{3} \nu \text{ grad div } \mathbf{V}. \quad (5)$$

The last term in equation (5) is a product of the viscosity kinematic coefficient and the value

$$\text{grad} \left(\frac{1}{\rho} \frac{d\rho}{dt} \right).$$

It equals zero in the case of inviscid or incompressible liquid. In the problem under consideration it may be supposed that viscosity is small and that compressibility is moderate, therefore this term can be neglected. The influence of the viscosity may be taken into consideration through boundary conditions. In spherical coordinates equation (5) will be

$$\frac{\partial v_r}{\partial t} + v_r \frac{\partial v_r}{\partial r} = -\frac{1}{\rho} \frac{\partial p}{\partial r}. \quad (6)$$

Integrating (6) allows us to obtain

$$\frac{\partial \phi}{\partial t} + \frac{v_r^2}{2} = - \int_{p_\infty}^p \frac{dp}{\rho} = -h(p). \quad (7)$$

It is assumed that $v_r = 0$ and $p = p_\infty$ at $r = \infty$.

Besides the equation (7) use is made of the Kirkwood and Bethe hypothesis [9] that

$$R^{5/2} = \frac{5}{2} \sqrt{\left(\frac{R_0^3}{a}\right)t},$$

where

$$a = \frac{3\rho_\infty}{2(p_\infty - p_v)}.$$

If the velocity of the bubble boundary is sufficiently high, equation (11) can be reduced to

$$\left(\frac{R_0}{R}\right)^3 \approx \left(\frac{\dot{R}}{3C}\right)^4 \frac{3\rho_\infty}{2(p_\infty - p_v)} \dot{R}^2 = \frac{\rho_\infty \dot{R}^6}{54(p_\infty - p_v)C^4}, \tag{15}$$

and then on the assumption $C \approx c_\infty$

$$\frac{R_0}{R} = \dot{R}^{2/3} \sqrt[3]{\left(\frac{\rho_\infty}{54(p_\infty - p_v)c_\infty^4}\right)}$$

or

$$\dot{R} = \sqrt{\left(\frac{R_0}{a}\right)} R^{-1/2}, \tag{16}$$

where

$$a = \sqrt[3]{\left(\frac{\rho_\infty}{54(p_\infty - p_v)c_\infty^4}\right)}, \tag{17}$$

equation (16) may be integrated easily:

$$\dot{R} = \sqrt{\left(\frac{R_0}{a}\right)} R^{-1/2} \quad \text{or} \quad \frac{dR}{dt} = \sqrt{\left(\frac{R_0}{a}\right)} R^{-1/2}$$

and

$$R = \left(\frac{3}{2}\right)^{2/3} \left(\frac{R_0}{a}\right)^{1/3} t^{2/3} = At^{2/3}, \tag{18}$$

where

$$A = 3 \times 32^{-1/9} R_0^{1/3} c_\infty^{4/9} (p_\infty - p_v)^{1/9} \rho_\infty^{-1/9}. \tag{19}$$

The quasi-acoustical approximation may be used for definition of velocities and pressures. According to the approximation made by Trilling [11], it is assumed that the velocity potential will satisfy the acoustical equation

$$\left[\frac{\partial}{\partial t} + c_\infty \frac{\partial}{\partial r}\right](r\phi) = 0, \tag{20}$$

and the momentum equation (1) can be written as

$$\frac{\partial v_r}{\partial t} + v_r \left(\frac{\partial v_r}{\partial r}\right) + \frac{1}{\rho} \left(\frac{\partial p}{\partial r}\right) = 0. \tag{21}$$

Taking into account that $v_r = (\partial\phi/\partial r)$ equation (21) may be integrated on variable r :

$$\frac{\partial\phi}{\partial t} + \frac{v_r^2}{2} + \int_{p_\infty}^p \frac{dp}{\rho} = 0. \tag{22}$$

Since

$$\left. \begin{aligned} v_r &= \frac{\partial\phi}{\partial r} \\ r\phi &= g\left(t - \frac{r-R}{c_\infty}\right) \end{aligned} \right\},$$

where g is an arbitrary function, and the local liquid velocity will be

$$v_r = -\frac{g\left(t - \frac{r-R}{c_\infty}\right)}{r^2} - \frac{g'\left(t - \frac{r-R}{c_\infty}\right)}{rc_\infty}. \tag{23}$$

If the density ρ will be changed by ρ_∞ in the first order approximation, then

$$h = \int_{p_\infty}^p \frac{dp}{\rho_\infty} = \frac{p-p_\infty}{\rho_\infty}$$

and on the basis of equation (7), the pressure may be expressed as

$$p = p_\infty - \frac{\rho_\infty g'\left(t - \frac{r-R}{c_\infty}\right)}{r} - \frac{\rho_\infty v_r^2}{2}. \tag{24}$$

The solution (23) and (24) in terms of an unknown function g and its derivative g' results

$$\left. \begin{aligned} g\left(t - \frac{r-R}{c_\infty}\right) &= \frac{r^2}{c_\infty} \left(c_\infty v_r - \frac{v_r^2}{2} - \frac{p-p_\infty}{\rho_\infty}\right) \\ g'\left(t - \frac{r-R}{c_\infty}\right) &= -r \left(\frac{v_r^2}{2} - \frac{p-p_\infty}{\rho_\infty}\right) \end{aligned} \right\}. \tag{25}$$

On the boundary of the bubble, where $r = R$, $v_r = \dot{R}$ and $p = p_v - (4\mu R/R)$ (neglecting surface tension), these functions will be

$$\left. \begin{aligned} g(t) &= -\frac{R^2}{c_\infty} \left(c_\infty \dot{R} - \frac{\dot{R}^2}{2} + \frac{p_\infty - p_v}{\rho_\infty} + \frac{4\mu \dot{R}}{\rho_\infty R}\right) \\ g'(t) &= -\frac{R\dot{R}^2}{2} + \frac{R(p_\infty - p_v)}{\rho_\infty} + \frac{4\mu \dot{R}}{\rho_\infty} \end{aligned} \right\}. \tag{26}$$

Now, substitution of the expressions (23) and (26) into (24) allows us to obtain

$$\begin{aligned} p - p_\infty = p_s &\approx -\frac{\rho_\infty}{r} g' - \frac{\rho_\infty}{2} \left[\frac{g}{r^2} + \frac{g'}{rc_\infty}\right]^2 \\ &\approx -\frac{\rho_\infty}{r} g' - \frac{\rho_\infty g'^2}{2r^2 c_\infty^2}. \end{aligned} \tag{27}$$

The last term in this expression can be omitted retaining only first order terms, since acoustical pressures are considered for great distances from a bubble. Then the adiabatic acoustical pressure is in the form

$$p_s \approx \frac{\rho_\infty}{r} \left[\frac{R\dot{R}^2}{2} - \frac{R(p_\infty - p_v)}{\rho_\infty} + \frac{4\mu \dot{R}}{\rho_\infty}\right]. \tag{28}$$

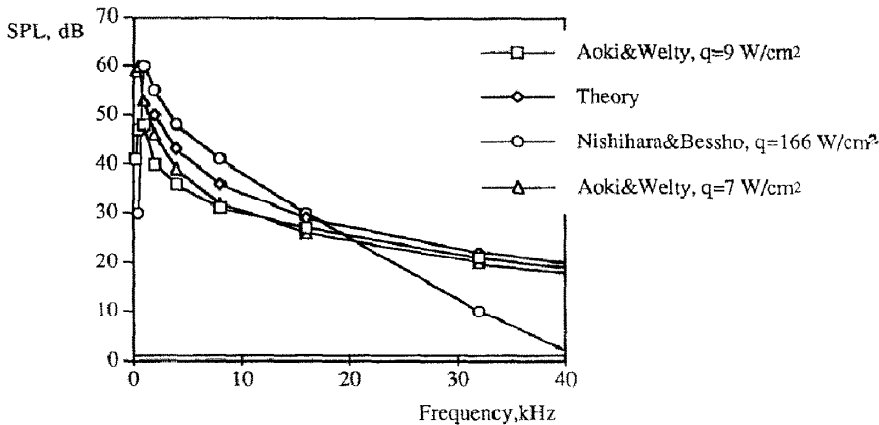


FIG. 1. Comparison of experimental and theoretical data.

Substitution of the expression (16) into (28) and exclusion of constant pressure components yield

$$p_s \approx -\frac{1}{r} \left[R(p_\infty - p_v) - 4\mu \sqrt{\left(\frac{R_0}{a}\right) R^{1/2}} \right]. \quad (29)$$

The total acoustical energy emitted by the bubble is

$$E = \frac{4\pi r^2}{\rho_\infty c_\infty} \int_0^t p_s^2 dt, \quad (30)$$

where (neglecting the second order infinitesimals)

$$p_s^2 \approx \frac{1}{r^2} \left[(p_\infty - p_v)^2 R^2 - 8\mu \sqrt{\left(\frac{R_0}{a}\right) R^{1/2}} \right]. \quad (31)$$

On the other hand, the emitted acoustical energy can be also defined as

$$E = \int_0^t S df, \quad (32)$$

where S is the spectral density of energy and f is the frequency.

Comparing the right hand sides of expressions (30) and (32) and taking into account that $f = 1/t$ allows us to obtain

$$\lim_{f \rightarrow \infty} \int_0^f S(f) df = -\frac{\rho_\infty c_\infty}{4\pi r^2} \left[(p_\infty - p_v)^2 A^2 f^{-7/3} - 8\mu \sqrt{\left(\frac{R_0}{a}\right) A^{1/2} f^{-4/3}} \right],$$

which after differentiating yields the expression for the spectral density

$$S(f) = \frac{\rho_\infty c_\infty}{4\pi r^2} \left[21 \times 2^{-10/9} R_0^{2/3} c_\infty^{8/9} \times (p_\infty - p_v)^{20/9} \rho_\infty^{-2/9} f^{-10/3} + 2^{44/9} \times \mu R_0^{2/3} c_\infty^{8/9} (p_\infty - p_v)^{2/9} \rho_\infty^{-2/9} f^{-7/3} \right]$$

or

$$S(f) = \frac{\rho_\infty^{7/9} c_\infty^{17/19} R_0^{2/3}}{4\pi r^2} \left[9.72 (p_\infty - p_v)^{20/9} f^{-10/3} + 29.5 \mu (p_\infty - p_v)^{2/9} f^{-7/3} \right], \quad (33)$$

where the equilibrium size of a vapor bubble R_0 is given by expression (13) and saturated vapor pressure $p_v = p_v(T_0)$ can be taken from ref. [12].

CONCLUSION

The expression for the high-frequency part of the noise spectrum obtained in this work differs from the results obtained in ref. [8], where the spectral density of the energy for boiling noise decreased proportionally $f^{-2/5}$. It follows from equation (33) that taking into account the liquid compressibility and viscosity gives the stronger dependence for spectral density vs the frequency, i.e. $S(f) \sim f^{-7/3}$, as the second term of the expression (33) will prevail at big frequency values. It may be seen that the falldown of the spectral density is approximately of $10 \log_{10} 2^{-7/3} = 7 \text{ dB octave}^{-1}$ in the region of high frequencies is in agreement with the results of Nishihara and Bessho [3]. The comparison of experimental and theoretical data is presented in Fig. 1. At the same time substituting the superheat $T - T_s$ into the expression for boiling spectra (33) yields that the boiling noise intensity varies as $(T - T_s)^{-2/3}$, i.e. decreases with an increase of the superheat in an agreement with experimental data of Aoki and Welty [2].

Acknowledgement—This study was supported in part by a grant 3531-1-91 from the Israeli Ministry of Science and Technology.

REFERENCES

1. A. B. Ponter and C. P. Haigh, Sound emission and heat transfer in low pressure pool boiling, *Int. J. Heat Mass Transfer* **12**, (1969).
2. T. Aoki and Y. R. Welty, Frequency distribution of boiling-generated sound, *J. Heat Transfer* (Aug. 1970).

3. H. Nishihara and Y. Bessho, Structure of acoustic noise spectra in subcooled pool boiling, ASME Publication, 74-WA/HT-31 (1974).
4. R. H. Mellen, Ultrasonic spectrum of cavitation noise in water, *JASA* **26**(3) (1954).
5. H. M. Fitzpatric, Cavitation noise, *Second Symposium on Naval Hydrodynamics* (1958).
6. T. Benjamin, Pressure waves from collapsing cavities, *Second Symposium on Naval Hydrodynamics* (1958).
7. F. R. Gilmore, The growth or collapse of a spherical bubble in a viscous compressible liquid, *Proceedings of the Heat Transfer and Fluid Mechanics Institute Held at the University of California at Los Angeles* (1952).
8. L. Likhterov and T. Elperin, High frequency asymptotics of the acoustic noise spectrum in nucleate pool boiling, *Int. Commun. Heat Mass Transfer* **17**(2) (1990).
9. J. G. Kirkwood and H. A. Bethe, The pressure wave produced by an underwater explosion, OSRD Report No. 588 (1942).
10. S. van Stralen and R. Cole, *Boiling Phenomena*, Vols. 1 and 2. McGraw-Hill, New York (1979).
11. L. Trilling, *J. Appl. Phys.* **23**(14), (1952).
12. G. J. van Wylen and R. E. Sonntag, *Fundamentals of Classical Thermodynamics* (3rd Edn). Wiley, New York (1985).